

**8.1** AISI 4340 steel (Fig. 8.31),  $\sigma_o = 800$  and 1600 MPa.

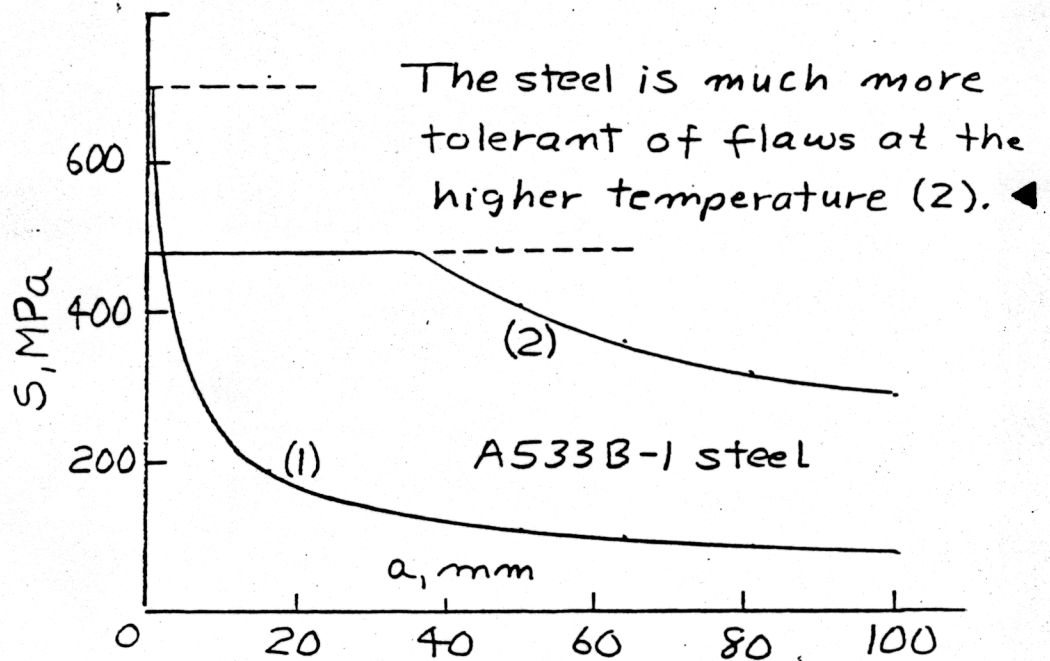
(a)  $K_{Ic} = ?$  (b)  $a_t = \frac{1}{\pi} \left( \frac{K_{Ic}}{\sigma_o} \right)^2$

$\sigma_o$ , MPa	$K_{Ic}$ , MPa $\sqrt{m}$	$a_t$ , mm
800	185	17.0
1600	70	0.20

The much smaller  $a_t$  for the higher  $\sigma_o$  indicates a greater sensitivity to flaws, so that brittle fracture would be an important design consideration.

8.2  $a_t = \frac{1}{\pi} \left( \frac{K_{Ic}}{\sigma_0} \right)^2$ ,  $K_{Ic} = S \sqrt{\pi a}$

Temp. °C	$K_{Ic}$ MPa√m	$\sigma_0$ MPa	$a_t$	
			m	mm
(1) -150	42	700	0.00115	1.15
(2) +10	160	480	0.0354	35.4



$$(1) S = \frac{K_{Ic}}{\sqrt{\pi a}} = \frac{42 \text{ MPa}\sqrt{\text{m}}}{\sqrt{\frac{\pi a \text{ mm}}{1000 \text{ mm/m}}}} = \frac{749}{\sqrt{a}} \text{ MPa}$$

$$(2) S = \frac{K_{Ic}}{\sqrt{\pi a}} = \frac{160}{\sqrt{\frac{\pi a}{1000}}} = \frac{2855}{\sqrt{a}} \text{ MPa}$$

**8.5**

Center-cracked plate, AISI 1144 steel,  $b = 40$ ,  $t = 15$  mm.  $K_{Ic} = 66 \text{ MPa}\sqrt{\text{m}}$  (Tbl. 8.1)

$P = ?$  for  $X = P_c / P = 3$  if  $a = 10, 24$  mm

(a)  $a = 10$  mm,  $\alpha = a/b = 10/40 = 0.25$

$$K = F S_y \sqrt{\pi a}, \quad F \approx 1 \quad (\text{Fig. 8.12})$$

$$S_y = \frac{P_c}{2bt} = \frac{XP}{2bt}, \quad P = \text{allowable design load}$$

$$K_{Ic} = F \frac{XP}{2bt} \sqrt{\pi a}$$

$$66 \text{ MPa}\sqrt{\text{m}} = \frac{3(P, \text{N})}{2(40)(15) \text{ mm}^2} \sqrt{\pi(0.010 \text{ m})}$$

$$P = 148,900 \text{ N} = 148.9 \text{ kN} \quad \blacktriangleleft$$

(b)  $a = 24$  mm,  $\alpha = a/b = 0.6$

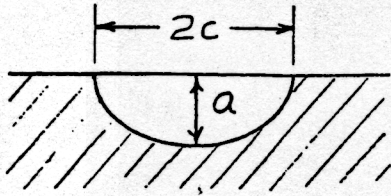
$$F = \frac{1 - 0.5\alpha + 0.326\alpha^2}{\sqrt{1-\alpha}} = 1.292 \quad (\text{Fig. 8.12})$$

$$66 = 1.292 \frac{3P}{2(40)(15)} \sqrt{\pi(0.024)}$$

$$P = 74,400 \text{ N} = 74.4 \text{ kN} \quad \blacktriangleleft$$

8.9

ASTM A470-8 steel,  $S = 250 \text{ MPa}$



$$2c = 50 \text{ mm}$$

$$a = 15 \text{ mm}$$

$$K_{Ic} = 60 \text{ MPa}\sqrt{\text{m}}$$

(Table 8.1)

Safe?

Fig. 8.19 applies

$$K = FS \sqrt{\frac{\pi a}{Q}}, \quad F = 1.12 \quad \left(\frac{a}{t}, \frac{c}{b} \text{ small}\right)$$

$$Q = 1 + 1.464 \left(\frac{a}{c}\right)^{1.65} = 1 + 1.464 \left(\frac{15 \text{ mm}}{25 \text{ mm}}\right)^{1.65}$$

$$Q = 1.630$$

$$K = 1.12 (250 \text{ MPa}) \sqrt{\frac{\pi (0.015 \text{ m})}{1.630}}$$

$$K = 47.6 \text{ MPa}\sqrt{\text{m}}$$

$$X = \frac{K_{Ic}}{K} = \frac{60 \text{ MPa}\sqrt{\text{m}}}{47.6 \text{ MPa}\sqrt{\text{m}}} = 1.26 \quad \text{No!}$$

$X$  is inadequate to safely operate.

**8.12** Shaft,  $d = 50$  mm, with circumferential crack,  $a = 5$  mm. 18 Ni maraging steel  
 $K_{Ic} = 123 \text{ MPa}\sqrt{\text{m}}$  (air melted, Table 8.1)

(a)  $\chi = ?$  if  $M = 1.5 \text{ kN}\cdot\text{m}$

Fig. 8.14 (b) applies:  $S_g = 4M/(\pi b^3)$

$$K = FS_g\sqrt{\pi a}, \quad \alpha = a/b, \quad \beta = 1 - \alpha$$

$$b = d/2 = 25 \text{ mm}, \quad \alpha = 5/25 = 0.2, \quad \beta = 0.8$$

$$F = \frac{3}{8\beta^{2.5}} \left[ 1 + \frac{\beta}{2} + \frac{3}{8}\beta^2 + \frac{5}{16}\beta^3 + \frac{35}{128}\beta^4 + 0.537\beta^5 \right]$$

$$F = 1.368$$

$$K = FS_g\sqrt{\pi a} = 1.368 \frac{4(1.5 \times 10^6 \text{ N}\cdot\text{mm})}{\pi (25 \text{ mm})^3} \sqrt{\pi (0.005 \text{ m})}$$

$$K = 20.95 \text{ MPa}\sqrt{\text{m}}, \quad \chi = K_{Ic}/K = 5.87 \quad \blacktriangleleft$$

(b) Add 120 kN tension (Fig. 8.14 (a))

$$F_2 = \frac{1}{2\beta^{1.5}} \left[ 1 + \frac{\beta}{2} + \frac{3}{8}\beta^2 - 0.363\beta^3 + 0.731\beta^4 \right] = 1.225$$

$$K_2 = F_2 S_{g2} \sqrt{\pi a}, \quad S_{g2} = P/(\pi b^2)$$

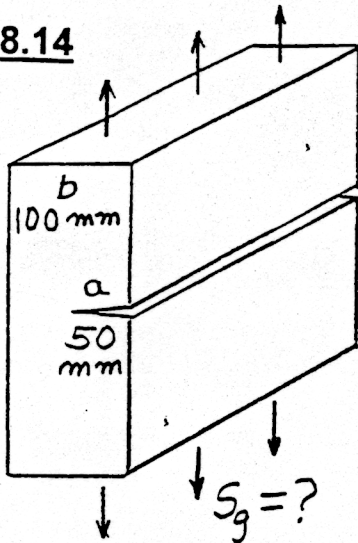
$$K_2 = 1.225 \frac{120,000 \text{ N}}{\pi (25 \text{ mm})^2} \sqrt{\pi (0.005 \text{ m})} = 9.39 \text{ MPa}\sqrt{\text{m}}$$

$$K = K_1 + K_2 = 20.95 + 9.39 = 30.33 \text{ MPa}\sqrt{\text{m}}$$

where  $K_1$  is  $K$  due to bending from (a).

$$\chi = K_{Ic}/K = 123/30.33 = 4.05 \quad \blacktriangleleft$$

8.14

A533B-1 steel,  $-75^\circ\text{C}$ 

$$K_{Ic} = 52 \text{ MPa}\sqrt{\text{m}}$$

$$\sigma_o = 550 \text{ MPa}$$

$$K = F S_g \sqrt{\pi a} = K_{Ic}$$

$$F = F(a/b = 0.5)$$

$F = 2.82$  from equation  
for Fig. 8.12(c)

$$S_g = \frac{K_{Ic}}{F\sqrt{\pi a}} = \frac{52 \text{ MPa}\sqrt{\text{m}}}{2.82\sqrt{\pi \times 0.05 \text{ m}}} = 46.5 \text{ MPa} \blacktriangleleft$$

Discussion: The usual basis of ductile weld design is to keep the net stress below yield by some safety factor  $X_o$ .

$$S_m = \frac{P}{(b-a)t} = \frac{P}{bt(1-a/b)} = \frac{S_g}{1-a/b} = \frac{46.5}{1-0.5} = 93 \text{ MPa}$$

$$X_o = \frac{\sigma_o}{S_m} = \frac{550}{93} = 5.9$$

This  $X_o$  suggests that there is considerable safety, but if the flaw acts as a crack, the safety factor that controls the behavior is  $X_K = 1$  at  $S_g = 46.5 \text{ MPa}$ .